

Thermal Performance Benchmarking



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Project ID: EDT070

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Overview

Timeline

- **Project start date:** FY15
- Project end date: FY17
- Percent complete: 15%

Budget

- Total project funding: \$200K
 DOE share: \$200K
- **Funding for FY15:** \$200K

Barriers

- Weight
- Performance and Lifetime
- Efficiency

Partners

- Oak Ridge National Laboratory (ORNL)
- Argonne National Laboratory (ANL)
- National Renewable Energy Laboratory (NREL) – Project Lead

Relevance: Objectives

Overall objective: To benchmark the thermal characteristics of the power electronics and electric motor thermal management systems

 Understand the current state-of-the-art (SOA) in thermal management systems and develop methods to improve on the SOA

FY15 objective: Evaluate the thermal management systems for the 2014 Honda Accord power electronics and the 2012 Nissan Leaf® power electronics and electric motor

Relevance: Impact

The information collected from these benchmarking activities will:

- Evaluate advantages and disadvantages of different thermal management systems
- Identify areas of improvement to advance the SOA
- Establish baseline metrics for the thermal management systems
- Increase the publicly available information related to automotive traction-drive thermal management systems
- Help guide future Electric Drive Technologies (EDT) R&D efforts
- Help industry to reduce the weight, volume, and cost of vehicle traction-drive systems by providing information that may influence future product designs
- Determine the operating temperatures for the EDT components in real-world operation.

Milestones

Month/ Year	Milestone or Go/No-Go Decision	Status
December 2014	Milestone: Draft the test plan for characterizing the thermal performance for both the inverter and electric motor.	Met
March 2015	Milestone: Modify the test loops (e.g., water-ethylene glycol [WEG] loop, automatic transmission fluid test loops) to accommodate the inverter and electric motor. Design and fabricate parts as required.	Met
June 2015	Milestone: Calibrate the various sensors (e.g., thermocouples, pressure transducers). Instrument the test articles (e.g., power module in the inverter) with temperature and pressure sensors for the experiments.	In progress
September 2015	Milestone : Complete thermal characterization of a commercially-available inverter and electric motor and summarize results in a report.	Upcoming
September 2015	Go/No-Go: Make a decision on the future power electronics and electric motor systems to be benchmarked.	Upcoming

Approach/Strategy

Collaborate with industry and ORNL to identify the vehicle system to benchmark

Acquire the vehicle components



- Experimentally measure thermal performance metrics
- Utilize modeling, particle image velocimetry, high speed video, and infrared imaging to understand heat transfer mechanisms

Analyze the data and calculate thermal performance metrics

Share results with industry and research institutions

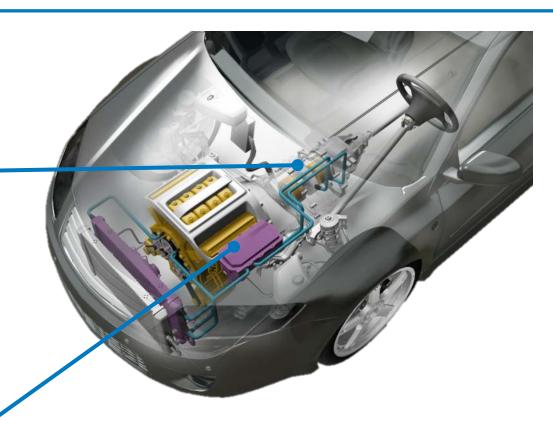
Approach/Strategy: Thermal Measurements

Electric motor thermal management

- Winding-to-liquid thermal resistance
- Motor lamination and winding thermal properties
- Pressure drop through the heat exchanger
- Volume and weight of the heat exchanger

Power electronics thermal management

- Junction-to-liquid thermal resistance
- Interface material thermal resistance
- Capacitor thermal properties
- Thermal resistance and pressure drop through the heat exchanger
- Volume and weight of the heat exchanger



Thermal management components

Pump pressure versus flow rate characteristics and efficiency

Approach/Strategy: Assumptions

 The experiments are designed to measure the performance of the motor and power electronic thermal management systems. This requires test procedures to accurately measure the heat dissipated by the components and the component temperatures. These test procedures do not replicate automotive environments or operating conditions. Strategies to improve thermal performance can then be deduced from these tests.

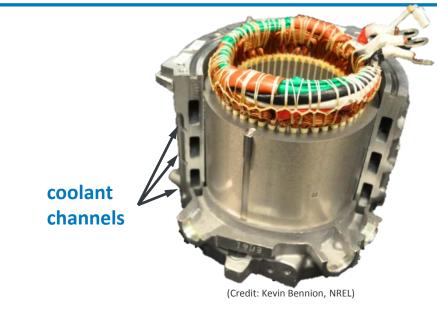
Approach/Strategy: FY15 Schedule

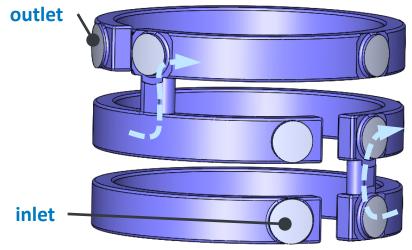
2014 Oct	Nov	Dec	2015 Jan	Feb	Mar	Apr	May	Jun	Jul	Aug	Sep
		manager	or therma nent syst lissan Le	em	man	inverter	system				
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Technical Accomplishments: Motor Thermal Management System

 Cast aluminum cooling jacket pressed around the stator

 WEG circulated through three cooling channels within the cooling jacket





View of the cooling channels showing the WEG flow path

Technical Accomplishments: Test Procedures

- Connected the motor to the WEG test bench
- Circulated WEG (50/50) at 65°C
 through the cooling jacket at
 different flow rates [2, 4, 8, 10, and
 12 liters per minute (lpm)]
- Heated the windings by running a high current (low voltage) through all phases
- Measured the motor temperature at various locations using thermocouples.



Motor (with insulation) connected to the WEG test bench

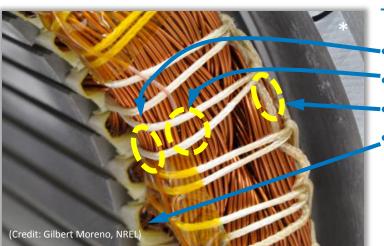


Thermal image of motor with all phases heated

Instrumented the motor with 40 thermocouples to measure temperatures and compute thermal resistances at various locations on the motor:

- Installed 20 thermocouples on end-winding surfaces
- Installed 10 thermocouples on the stator and slot liner
- Installed 10 thermocouples on the cooling jacket.

Installed 20 thermocouples on end-winding surfaces



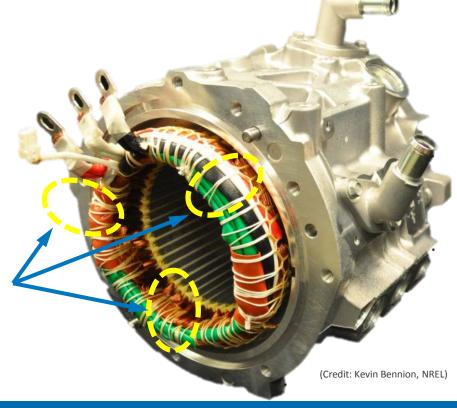
*Note: Image shows thermocouples prior to being bonded to the motor with thermally-conductive epoxy

Thermocouples were installed on

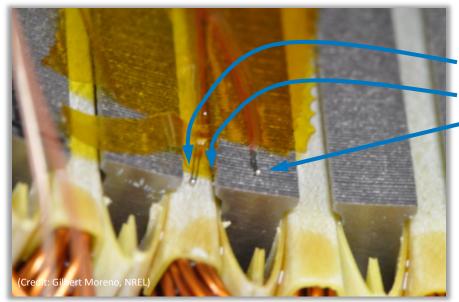
- inside-winding surface
- top-winding surface
- outside-winding surface

between end-winding and slot liner.

Thermocouples were installed at three locations equally spaced on the end windings (both sides).



Installed 10 thermocouples on the stator and slot-winding surfaces

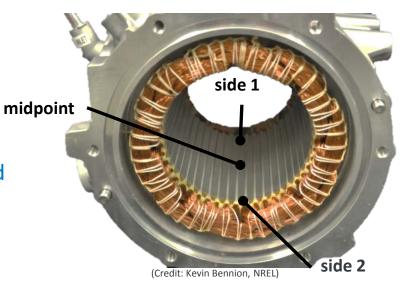


*Note: Image shows thermocouples prior to being bonded to the motor with thermally-conductive epoxy

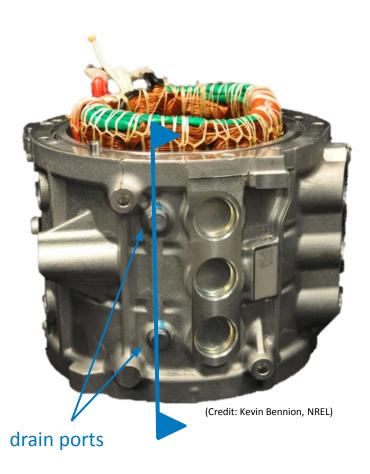
Thermocouples were installed on

- Slot-liner surface
- Between stator lamination and slot liner
- Stator-lamination inside surface.

Thermocouples were installed on both sides and midpoint of the motor to evaluate temperature variations in the axial direction.

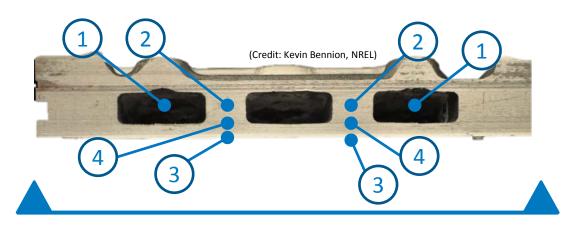


Installed 10 thermocouples on the cooling jacket



Thermocouples installed on (bullet numbers coincide with numbers in figure below)

- 1. Drain ports and on the inlet and outlet (WEG temperatures)
- 2. Midpoint between coolant channels
- 3. Stator and cooling jacket interface
- 4. Cooling jacket, 2 mm above stator surface.



Cross-sectional view of the cooling jacket

Technical Accomplishments: Energy Balance

Confirmed that the majority of the heat is absorbed by the WEG via the cooling jacket

Used the heat absorbed by the WEG for the thermal resistance

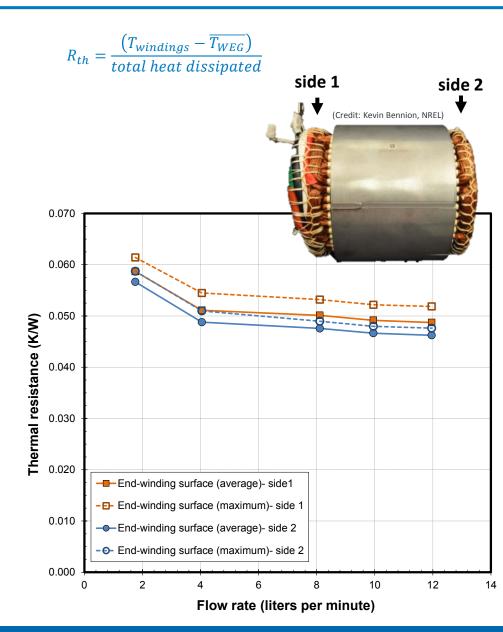
calculations.

$$m{Q} = m{m} \ m{C_p} \ (m{T_{out}} - m{T_{in}})$$
 $m{m} = ext{mass flow rate} \ m{C_p} = ext{specific heat} \ m{Q} = ext{heat} \ m{T_{out}} = ext{outlet WEG temperature} \ m{T_{in}} = ext{inlet WEG temperature}$

WEG flow rate	Total heat input	Heat absorbed by WEG	Percent of total heat absorbed by		
rate	mpac	by WEG	WEG		
lpm	watts	watts	%		
1.8	564.7	508.8	90.1%		
4.0	565.6	529.9	93.7%		
8.1	566.8	536.9	94.7%		
10.0	567.8	543.0	95.6%		
12.0	567.3	542.4	95.6%		

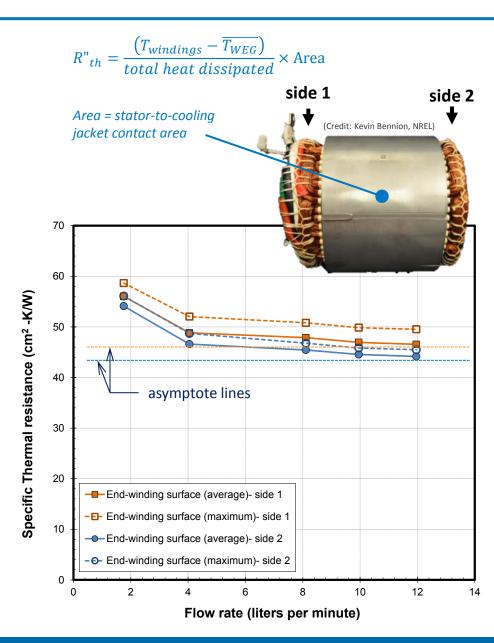
Technical Accomplishments: End-Winding Thermal Resistance

- Higher thermal resistance on the electrical-connection side (side 1) of the motor is a result of the heat generated by the electrical cables.
- Increasing the WEG flow rates has minimal effect on the thermal resistances. This behavior indicates that the passive stack thermal resistance is significantly greater than the convective thermal resistance.
- The high passive stack thermal resistance means that increasing the WEG flow rates beyond ~4 lpm has minimal effect on decreasing the thermal resistances. Improving the thermal performance of this WEG-cooled motor requires a significant reduction in the motor passive stack thermal resistance.

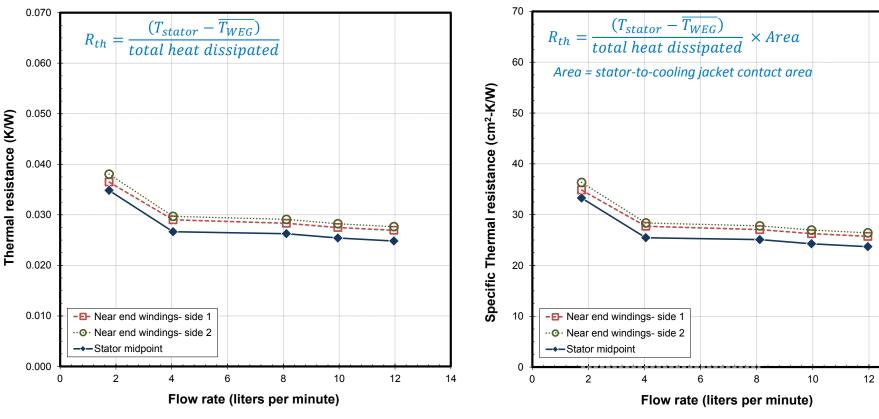


Technical Accomplishments: End-Winding Thermal Resistance

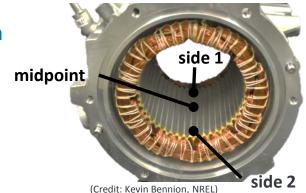
- Scaled the thermal resistances by the stator-to-cooling jacket contact area.
 Scaling the thermal resistances provides a metric that can be used when comparing the thermal performance of different motors
- The thermal resistance curve asymptotic lines indicate the point when $R_{th, \text{ passive stack}} >> R_{th, \text{ convective.}}$ Therefore, the thermal resistance asymptote lines can be used to estimate the passive stack thermal resistances.



Technical Accomplishments: Stator Thermal Resistance

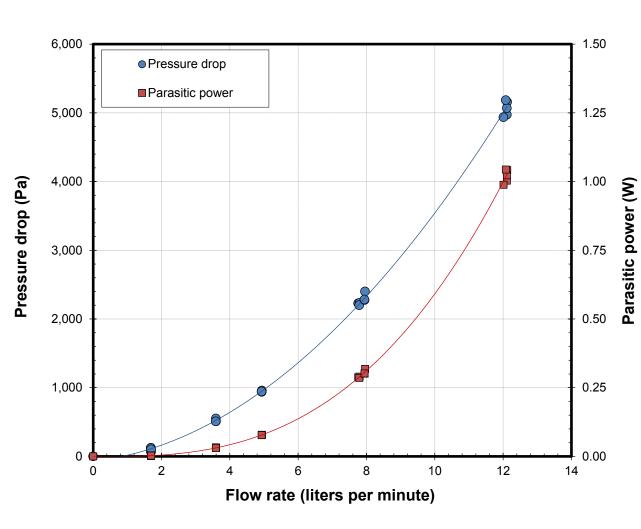


- Stator-to-liquid thermal resistances are essentially equal on both sides of the motor and are lowest at the motor midpoint
- Stator-to-liquid thermal resistances are lower than the windingto-liquid (previous slides) thermal resistances. This effect is the results of a shorter heat-flow path from the inside of the stator to the coolant.



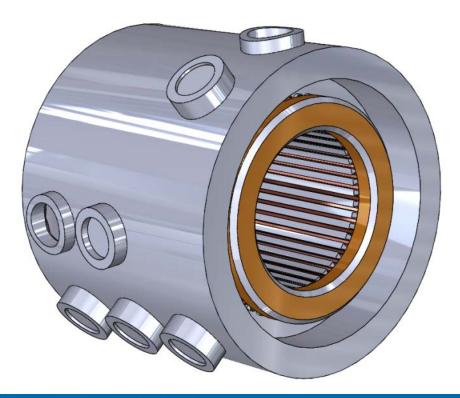
Technical Accomplishments: Pressure Drop and Parasitic Power

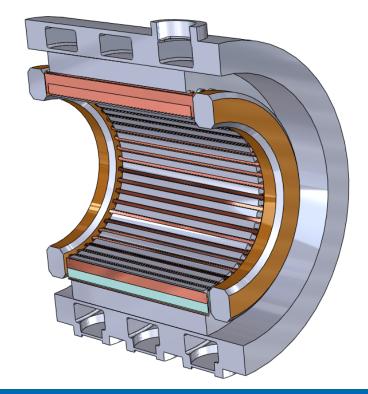
- Measured the pressure drop and computed the parasitic power through the cooling jacket
- Parasitic power will be used to compute the system efficiency (e.g., coefficient of performance) metrics.



Technical Accomplishments: Motor CAD Drawing

- Generated computer-aided design (CAD) drawings of the motor
- Used for finite element (FE) and computational fluid dynamics (CFD) simulations
- FE and CFD analysis combined with experimental results will allow us to quantify the various thermal resistances within the motor (e.g., cooling jacket thermal resistance)





Responses to Previous Year Reviewers' Comments

This is a new project. It was not reviewed in FY14.

Collaboration and Coordination with Other Institutions

Government laboratories

- Oak Ridge National Laboratory
- Argonne National Laboratory

Remaining Challenges and Barriers

 Challenge: Use the data from these experiments to quantify/understand the temperatures experienced by these components during actual, onthe-road driving conditions.

Proposed Future Work

FY15

- Complete testing of the 2012 Nissan Leaf motor thermal management system.
- Characterize and identify methods to improve thermal performance of the 2012 Nissan Leaf and 2014 Honda Accord inverter thermal management systems.

FY16

 Characterize and identify methods to improve the thermal performance of the 2014 Honda Accord motor (oil-cooled system) and 2015 BMW i3 motor and power electronics thermal management systems.

Summary

Relevance

• This work will increase the understanding of the current SOA in motor and power electronics thermal management systems and develop methods to improve on the SOA.

Approach

- Collaborate with industry and ORNL to identify the appropriate vehicle to benchmark
- Characterize the thermal performance of the inverter and motor thermal management systems and share the results with industry
- Identify areas of improvement to advance the SOA and establish baseline metrics for the thermal management systems.

Accomplishments

Initiated characterization of the 2012 Nissan Leaf motor thermal management system.

Collaborations

- Oak Ridge National Laboratory
- Argonne National Laboratory



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